## **LETTER TO THE EDITORS**

## **COMMENTS ON "PARALLEL FLOW AND COUNTER FLOW CONDENSATION ON AN INTERNALLY COOLED VERTICAL TUBE"**

IN **THE** above paper Sparrow and Faghri [lnt. J. Heat *Mass*  Transfer 23, 559-562 (1980)], it was mentioned that the following equation has to be solved numerically for parallel flow :

$$
\frac{\mathrm{d}\theta}{\mathrm{d}X} = \frac{1-\theta}{1+\beta\theta^{1/3}}.\tag{1}
$$

This equation however can be solved by integration by parts to yield an exact solution as given below :

$$
\frac{(1+\beta\theta^{1/3})}{(1-\theta)}\,\mathrm{d}\theta=\mathrm{d}X\tag{2}
$$

Integration of equation (2) with the condition  $\theta = 0$  at  $X = 0$ gives

$$
X = -\ln(1 - \theta) - 3\beta \left\{ \theta^{1/3} + \frac{1}{6} \ln \left[ \frac{(1 - \theta^{1/3})^2}{1 + \theta^{1/3} + \theta^{2/3}} \right] - \frac{1}{\sqrt{3}} \tan^{-1} \left( \frac{2\theta^{1/3} + 1}{\sqrt{3}} \right) + \frac{\pi}{6\sqrt{3}} \right\}.
$$
 (3)

Similarly the equation for counterflow case will also yield an

exact solution for X in terms of  $\beta$ ,  $\theta$  and  $\theta_e$  as given below:

$$
X = \ln\left[\frac{(1-\theta)}{(1-\theta_e)}\right] + 3\beta \left[y - \frac{a}{6}\ln\left\{\frac{(a+y)^2}{a^2 - ay + y^2}\right\}\right]
$$

$$
-\frac{a}{\sqrt{3}}\tan^{-1}\left(\frac{2y-a}{a\sqrt{3}}\right) - \frac{\pi a}{6\sqrt{3}}\right]
$$

where

$$
y = (\theta_e - \theta)^{1/3}
$$
 and  $a = (1 - \theta_e)^{1/3}$ . (4)

Thus the above two cases of condensation on vertical tubes can be solved exactly, thus avoiding the possible errors obtained with the numerical solution. The plots of  $\theta$  vs X in terms of  $\beta$  as given in Fig. 2 of the paper under discussion can be drawn using equation (3) given above.

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